



THE PAPUA NEW GUINEA UNIVERSITY OF TECHNOLOGY  
DEPARTMENT OF CIVIL ENGINEERING - 2<sup>ND</sup> YEAR DEGREE

SECOND SEMESTER EXAMINATIONS - 2021

CE 222 – STRUCTURAL ANALYSIS EXAM

DATE: FRIDAY, 29<sup>TH</sup> OCTOBER 2021 – 08:20 A.M.

VENUE: STRUCTURES LECTURE THEATRE (SLT)

TIME ALLOWED: 3 HOURS

INFORMATION FOR CANDIDATES

1. You have 10 minutes to read the paper before the examination starts. You must **not** begin writing during this time.
2. **There are Eight (8) Questions in this paper. Answer any FIVE (5) of the eight questions.**
3. Use only ink. Do not use pencils for writing except for drawings and sketches.
4. Start each question on a new page and show all your calculations in the answer book provided. No other material will be accepted.
5. Write on **one side of the page only** and keep the margins clear.
6. **Write your NAME and Student NUMBER clearly on the front page. Do it now.**
7. Calculators are permitted in the examination. Note and textbooks are not allowed.
8. Marks for each of the question are given within parenthesis at the end of each question.
9. **Switch your mobile phone OFF.**

**Question One [Integration Method]**

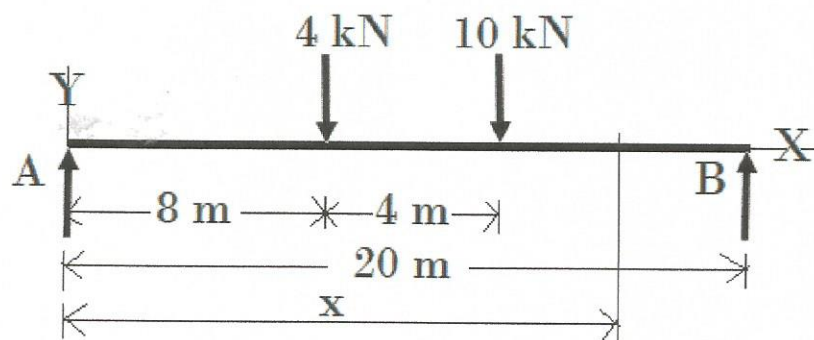
A simply supported beam shown in **Figure Q1A** is of span 20 m and carries two concentrated loads **4 kN** at 8 m and **10 kN** at 12 m from one end.

Calculate;

(a) the deflection under each load, (6 marks)

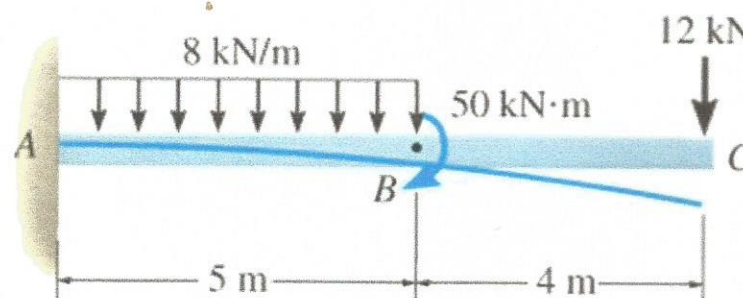
(b) the maximum deflection. (4 marks)

$$E = 200,000 \text{ N/mm}^2, I = 10^9 \text{ mm}^4$$



**Figure Q1A**

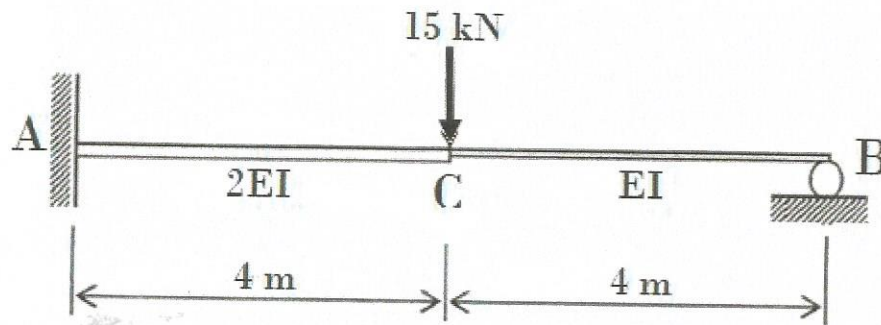
(c) Determine the equation for the elastic curve for the cantilevered beam in **Figure Q1B**.  $EI$  is constant. (10 marks)



**Figure Q1B**

**Question Two** [Moment-Area Method]

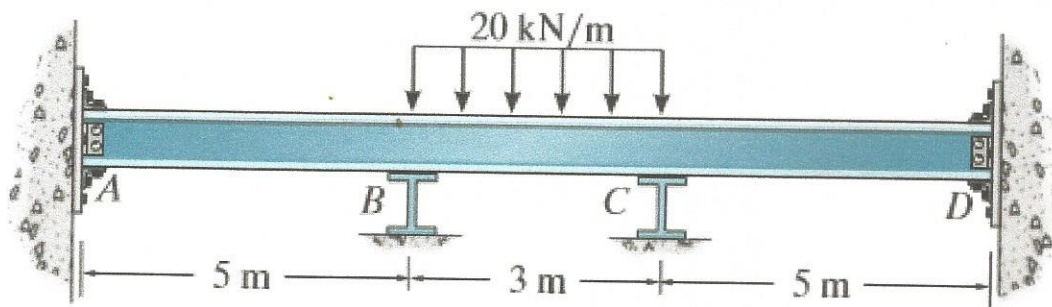
Using the **Moment-Area Method**, determine the slopes at **B** and **C** for the mild steel beam shown in **Figure Q2**, and also the deflection at the load point **C**. (20 marks)



**Figure Q2**

**Question Three** [Slope Deflection Method – Beam]

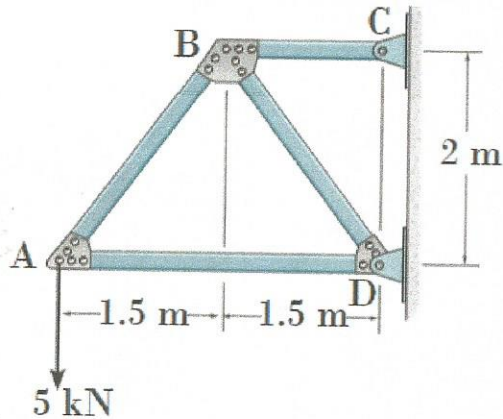
Use the **Slope Deflection Method** to determine the moments at A, B, C and D for the steel beam shown in **Figure Q3** below and plot the shear force and bending moment diagrams for the beam. Assume **EI** is constant. (20 marks)



**Figure Q3**

**Question Four [Energy Methods: Virtual Work/Castigliano's Theorem]**

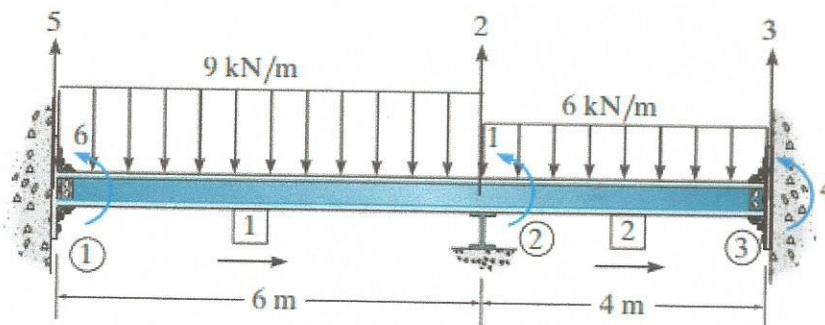
- (a) Determine the vertical displacement at joint **A** for the truss shown in **Figure Q4**. Each bar is made of steel and has a cross-sectional area of **600 mm<sup>2</sup>**. Take **E = 200 GPa**. Use the **Method of Virtual Work**. **EI** is constant. (10 marks)
- (b) Solve (a) using **Castigliano's theorem**. (10 marks)



**Figure Q4**

**Question Five Stiffness Matrix Method – Beam**

- Determine the reactions at the supports shown in **Figure Q5**. Assume 1 and 3 are fixed and 2 is a roller. **EI** is constant. (20 marks)



**Figure Q5**

## Question Six

## Moment-Area/ Conjugate Beam Method

- (a) Determine the slope and the displacement at the end **C** of the beam in **Figure Q6** using the Moment-Area Method.  $E = 200 \text{ GPa}$ ,  $I = 70 \times 10^6 \text{ mm}^4$ . (10 marks)
- (b) Solve the problem in (a) using the conjugate-beam method. (10 marks)

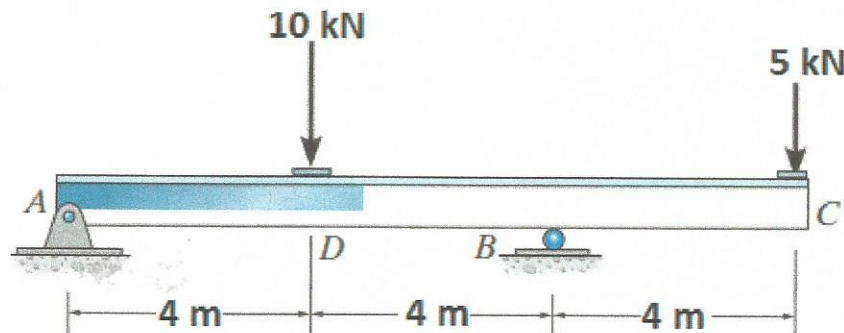


Figure Q6

## Question Seven

## Slope Deflection Method - Frame

Using the Slope Deflection Method, determine the reactions at the supports and the moment at **B** for the frame shown in **Figure Q7**, then draw the moment diagram for each member of the frame. Assume the support at **A** is fixed and **C** is pinned.

$EI$  is constant.

(20 marks)

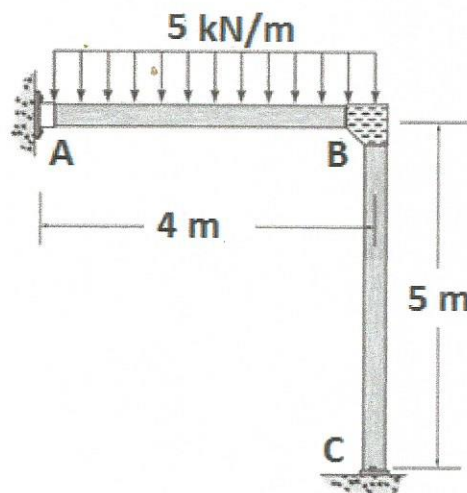


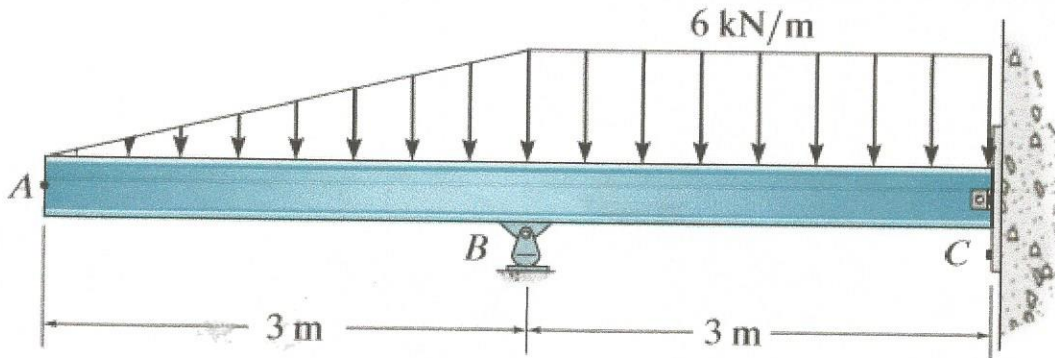
Figure Q7

## Question Eight

[Energy Methods: Virtual Work/Castigliano's Theorem]

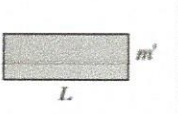
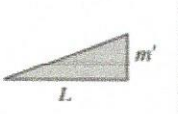

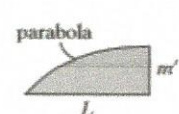
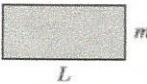
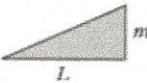
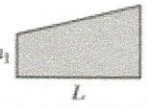
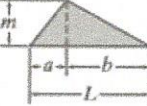
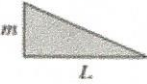
(a) Determine the slope and displacement at point **A** for the beam shown in **Figure Q8**. Use the principle of virtual work.  $EI$  is constant. (10 marks)

(b) Solve the problem in (a) using Castigliano's theorem. (10 marks)

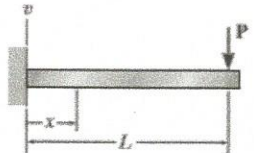
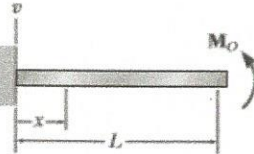
**Figure Q8**

**END OF EXAMINATION! ALL THE BEST!**

Table for Evaluating  $\int_0^L m m' dx$

$\int_0^L m m' dx$				
	$mm'L$	$\frac{1}{2}mm'L$	$\frac{1}{2}m(m'_1 + m'_2)L$	$\frac{2}{3}mm'L$
	$\frac{1}{2}mm'L$	$\frac{1}{3}mm'L$	$\frac{1}{6}m(m'_1 + 2m'_2)L$	$\frac{5}{12}mm'L$
	$\frac{1}{2}m'(m_1 + m_2)L$	$\frac{1}{6}m'(m_1 + 2m_2)L$	$\frac{1}{6}\{m'_1(2m_1 + m_2) + m'_2(m_1 + 2m_2)\}L$	$\frac{1}{12}\{m'(3m_1 + 5m_2)\}L$
	$\frac{1}{2}mm'L$	$\frac{1}{6}mm'(L + a)$	$\frac{1}{6}m\{m'_1(L + b) + m'_2(L + a)\}$	$\frac{1}{12}mm'\left(3 + \frac{3a}{L} - \frac{a^2}{L^2}\right)L$
	$\frac{1}{2}mm'L$	$\frac{1}{6}mm'L$	$\frac{1}{6}m(2m'_1 + m'_2)L$	$\frac{1}{4}mm'L$

Beam Deflections and Slopes

Loading	$v \uparrow$	$\theta \curvearrowright$	Equation $\uparrow \curvearrowright$
	$v_{\max} = \frac{PL^3}{3EI}$ at $x = L$	$\theta_{\max} = \frac{PL^2}{2EI}$ at $x = L$	$v = \frac{P}{6EI}(x^3 - 3Lx^2)$
	$v_{\max} = \frac{M_0L^2}{2EI}$ at $x = L$	$\theta_{\max} = \frac{M_0L}{EI}$ at $x = L$	$v = \frac{M_0}{2EI}x^2$

Beam Deflections and Slopes (continued)

	$v_{\max} = -\frac{wL^4}{8EI}$ at $x = L$	$\theta_{\max} = -\frac{wL^3}{6EI}$ at $x = L$	$v = -\frac{w}{24EI}(x^4 - 4Lx^3 + 6L^2x^2)$
	$v_{\max} = \frac{PL^3}{48EI}$ at $x = L/2$	$\theta_{\max} = \pm \frac{PL^2}{16EI}$ at $x = 0$ or $x = L$	$v = \frac{P}{48EI}(4x^3 - 3L^2x)$ $0 \leq x \leq L/2$
		$\theta_L = \frac{Pab(L+b)}{6LEI}$ $\theta_R = \frac{Pab(L+a)}{6LEI}$	$v = -\frac{Pbx}{6LEI}(L^2 - b^2 - x^2)$ $0 \leq x \leq a$
	$v_{\max} = -\frac{5wL^4}{384EI}$ at $x = \frac{L}{2}$	$\theta_{\max} = \pm \frac{wL^3}{24EI}$	$v = -\frac{wx}{24EI}(x^3 - 2Lx^2 + L^3)$
		$\theta_L = -\frac{3wL^3}{128EI}$ $\theta_R = \frac{7wL^3}{384EI}$	$v = \frac{wx}{384EI}(16x^3 - 24Lx^2 + 9L^3)$ $0 \leq x \leq L/2$ $v = \frac{wL}{384EI}(8x^3 - 24Lx^2 + 17L^2x - L^3)$ $L/2 \leq x \leq L$
	$v_{\max} = \frac{M_O L^2}{9\sqrt{3}EI}$	$\theta_L = \frac{M_O L}{6EI}$ $\theta_R = \frac{M_O L}{3EI}$	$v = -\frac{M_O x}{6EI}(L^2 - x^2)$

$$M_N = 2Ek(2\theta_N + \theta_F - 3\psi) + (FEM)_N$$

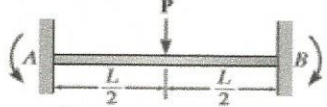
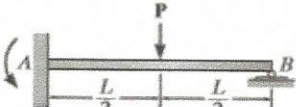
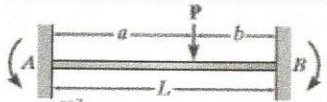
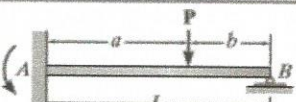
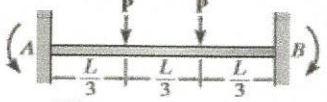
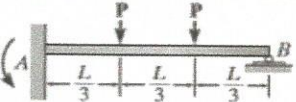
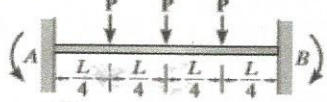
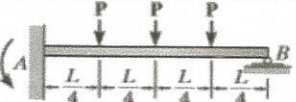
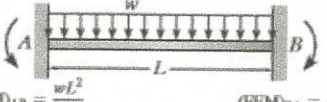
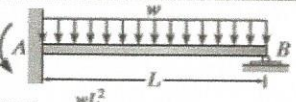
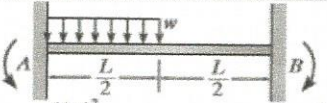
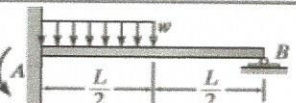
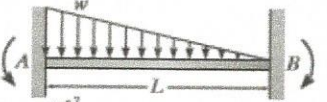
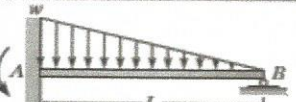
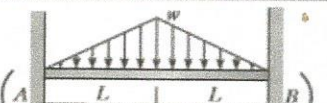
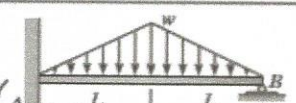
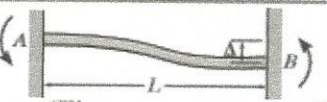
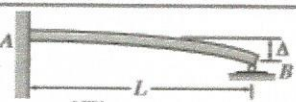
For Internal Span or End Span with Far End Fixed

$$M_N = 3Ek(\theta_N - \psi) + (FEM)_N$$

Only for End Span with Far End Pinned or Roller Supported



Fixed End Moments

 <p> <math>(FEM)_{AB} = \frac{PL}{8}</math> <math>(FEM)_{BA} = \frac{PL}{8}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{3PL}{16}</math> </p>
 <p> <math>(FEM)_{AB} = \frac{Pb^2a}{L^3}</math> <math>(FEM)_{BA} = \frac{Pa^2b}{L^3}</math> </p>	 <p> <math>(FEM)'_{AB} = \left(\frac{P}{L^3}\right)\left(b^2a + \frac{a^2b}{2}\right)</math> </p>
 <p> <math>(FEM)_{AB} = \frac{2PL}{9}</math> <math>(FEM)_{BA} = \frac{2PL}{9}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{PL}{3}</math> </p>
 <p> <math>(FEM)_{AB} = \frac{5PL}{16}</math> <math>(FEM)_{BA} = \frac{5PL}{16}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{15PL}{32}</math> </p>
 <p> <math>(FEM)_{AB} = \frac{wL^2}{12}</math> <math>(FEM)_{BA} = \frac{wL^2}{12}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{wL^2}{8}</math> </p>
 <p> <math>(FEM)_{AB} = \frac{11wL^2}{192}</math> <math>(FEM)_{BA} = \frac{5wL^2}{192}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{9wL^2}{128}</math> </p>
 <p> <math>(FEM)_{AB} = \frac{wL^2}{20}</math> <math>(FEM)_{BA} = \frac{wL^2}{30}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{wL^2}{15}</math> </p>
 <p> <math>(FEM)_{AB} = \frac{5wL^2}{96}</math> <math>(FEM)_{BA} = \frac{5wL^2}{96}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{5wL^2}{64}</math> </p>
 <p> <math>(FEM)_{AB} = \frac{6EI\Delta}{L^2}</math> <math>(FEM)_{BA} = \frac{6EI\Delta}{L^2}</math> </p>	 <p> <math>(FEM)'_{AB} = \frac{3EI\Delta}{L^2}</math> </p>

## Member Global Stiffness Matrix

$$q = k'TD$$

$$Q = T^T k'TD$$

$$Q = kD$$

$$k = T^T k'T$$

$$k = \begin{bmatrix} \lambda_x & 0 \\ \lambda_y & 0 \\ 0 & \lambda_x \\ 0 & \lambda_y \end{bmatrix} \frac{AE}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} \lambda_x & \lambda_y & 0 & 0 \\ 0 & 0 & \lambda_x & \lambda_y \end{bmatrix}$$

$$k = \frac{AE}{L} \begin{bmatrix} N_x & N_y & F_x & F_y \\ \lambda_x^2 & \lambda_x \lambda_y & -\lambda_x^2 & -\lambda_x \lambda_y \\ \lambda_x \lambda_y & \lambda_y^2 & -\lambda_x \lambda_y & -\lambda_y^2 \\ -\lambda_x^2 & -\lambda_x \lambda_y & \lambda_x^2 & \lambda_x \lambda_y \\ -\lambda_x \lambda_y & -\lambda_y^2 & \lambda_x \lambda_y & \lambda_y^2 \end{bmatrix} \begin{bmatrix} N_x \\ N_y \\ F_x \\ F_y \end{bmatrix}$$

**Beam Analysis Using Stiffness Method**

$$\begin{bmatrix} q_{Ny'} \\ q_{Nz'} \\ q_{Fy'} \\ q_{Fz'} \end{bmatrix} = \begin{bmatrix} \frac{12EI}{L^3} & \frac{6EI}{L^2} & -\frac{12EI}{L^3} & \frac{6EI}{L^2} \\ \frac{6EI}{L^2} & \frac{4EI}{L} & -\frac{6EI}{L^2} & \frac{2EI}{L} \\ -\frac{12EI}{L^3} & -\frac{6EI}{L^2} & \frac{12EI}{L^3} & -\frac{6EI}{L^2} \\ \frac{6EI}{L^2} & \frac{2EI}{L} & -\frac{6EI}{L^2} & \frac{4EI}{L} \end{bmatrix} \begin{bmatrix} d_{Ny'} \\ d_{Nz'} \\ d_{Fy'} \\ d_{Fz'} \end{bmatrix}$$

These equations can also be written in abbreviated form as

$$q = kd$$

**Structure Stiffness Equation**

$$Q = KD$$

**Q** and **D** are column matrices that represent both the known and unknown loads and displacements. Partitioning the stiffness matrix into the known and unknown elements of load and displacement, we have

$$\begin{bmatrix} Q_k \\ Q_u \end{bmatrix} = \begin{bmatrix} K_{11} & K_{12} \\ K_{21} & K_{22} \end{bmatrix} \begin{bmatrix} D_u \\ D_k \end{bmatrix}$$

which when expanded yields the two equations

$$Q_k = K_{11}D_u + K_{12}D_k$$

$$Q_u = K_{21}D_u + K_{22}D_k$$